

# COUPLING SELECTION ACC. TO DIN 740 PART II WITH SPECIFIC FACTORS

## Coupling types

### Backlash-free servo couplings

 <p><b>ROTEX® GS</b></p>	<p>Backlash-free, flexible jaw coupling (see page 116)</p> <ul style="list-style-type: none"> <li>- Backlash-free and flexible</li> <li>- Maintenance-free</li> <li>- Fail-safe</li> <li>- Compact dimensions, high power density</li> <li>- Single-cardanic or double-cardanic</li> <li>- Axial plug-in</li> <li>- High speeds</li> </ul>
 <p><b>TOOLFLEX®</b></p>	<p>Backlash-free, torsionally rigid metal bellow-type coupling (see page 116)</p> <ul style="list-style-type: none"> <li>- Backlash-free and torsionally rigid</li> <li>- Maintenance-free</li> <li>- Shear type</li> <li>- Compact dimensions</li> <li>- Double-cardanic</li> <li>- Axial plug-in (as an option)</li> <li>- All-steel</li> </ul>
 <p><b>RADEX®-NC</b></p>	<p>Backlash-free, torsionally rigid servo lamina coupling (see page 116)</p> <ul style="list-style-type: none"> <li>- Backlash-free and torsionally rigid</li> <li>- Maintenance-free</li> <li>- Compact dimensions</li> <li>- Single-cardanic or double-cardanic</li> <li>- All-steel</li> </ul>
 <p><b>COUNTEX®</b></p>	<p>Backlash-free, torsionally rigid shaft encoder coupling (see page 116)</p> <ul style="list-style-type: none"> <li>- Backlash-free and torsionally rigid</li> <li>- Maintenance-free</li> <li>- Compact dimensions</li> <li>- Double-cardanic</li> <li>- Axial plug-in</li> </ul>

## Terminology of coupling selection

Description	Symbol	Definition or explanation
Rated torque of coupling [Nm]	$T_{KN}$	Torque which can be transmitted continuously over the entire permissible speed range, taking into account the factors.
Maximum torque of coupling [Nm]	$T_{K \max.}$	Torque which can be transmitted over the entire service life of the coupling as dynamic load $\geq 10^6$ or as alternating load $5 \cdot 10^4$ taking into account the factors.
Rated torque of machine [Nm]	$T_N$	Stationary rated torque on the coupling
Rated torque of driving side [Nm]	$T_{AN}$	Constantly occurring driving torque as per the data indicated by the motor manufacturer
Peak torque [Nm]	$T_S$	Peak torque on the coupling
Peak torque of driving side [Nm]	$T_{AS}$	Peak torque with torque shock on driving side, e. g. starting torque of the servo motor as per the data indicated by the motor manufacturer.
Peak torque of load side [Nm]	$T_{LS}$	Peak torque with torque shock on load side, e. g. braking
Screw tightening torque [Nm]	$T_A$	Screw tightening torque
Friction torque [Nm]	$T_R$	Torque that can be transmitted through the frictionally engaged shaft-hub-connection

Description	Symbol	Definition or explanation
Rotational inertia coefficient of driving side	$M_A$	Factor taking into account the mass distribution with shocks and vibrations produced on the driving or load side.
Rotational inertia coefficient of load side	$M_L$	
Mass moment of inertia of driving side [kgm <sup>2</sup> ]	$J_A$	Total of moments of inertia existing on the driving or load side referring to the coupling speed.
Mass moment of inertia of load side [kgm <sup>2</sup> ]	$J_L$	
Mass moment of inertia of coupling [kgm <sup>2</sup> ]	$J_{KA}$	Mass mom. of inertia of the coupl. half on the drive side
	$J_{KL}$	Mass mom. of inertia of the coupl. half on the load side
Mass moment of inertia [kgm <sup>2</sup> ]	$J_{Mot}/J_{Sp}/J_{HS}$	Mass moment of inertia of the motor/mass moment of inertia of the spindle/mass moment of inertia of the main spindle
Shock factor on driving side	$S_A$	Factor taking into account the shocks arising depending on the application (e. g. starting shocks) With positioning drives the additional load is considered by the starting frequency per hour.
Shock factor on load side	$S_L$	
Temperature factor	$S_t$	Temperature factor - Factor considering the lower loading capacity or larger deformation of an elastomer part under load particularly with increased temperatures.
Operating factor	$S_B$	Factor considering the different demands on the coupling dependent on the application.

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## Factors

Temperature factor $S_t$														
	-50 °C	-40 °C	-30 °C	-20 °C/ +30 °C	≤ +40 °C	≤ +50 °C	≤ +60 °C	≤ +70 °C	≤ +80 °C	≤ +90 °C	≤ +100 °C	≤ +110 °C	≤ +120 °C	≤ +200 °C
<b>ROTEX® GS</b>														
Polyurethan 80 Sh-A-GS	1,0	1,0	1,0	1,0	1,2	1,3	1,4	1,55	1,8	–	–	–	–	–
Polyurethan 92 Sh-A-GS	–	1,0	1,0	1,0	1,2	1,3	1,4	1,55	1,8	2,2	–	–	–	–
Polyurethan 98 Sh-A-GS	–	–	1,0	1,0	1,2	1,3	1,4	1,55	1,8	2,2	–	–	–	–
Polyurethan 64 Sh-D-GS	–	–	–	1,0	1,2	1,3	1,4	1,55	1,8	2,2	3,0	–	–	–
Polyurethan 72 Sh-D-GS	–	–	–	1,0	1,2	1,3	1,4	1,55	1,8	2,2	3,0	–	–	–
Hytrel 64 Sh-D-H-GS	1,0	1,0	1,0	1,0	1,2	1,3	1,4	1,5	1,6	1,8	2,0	2,3	2,8	–
Hytrel 72 Sh-D-H-GS	1,0	1,0	1,0	1,0	1,2	1,3	1,4	1,5	1,6	1,8	2,0	2,3	2,8	–
<b>TOOLFLEX®</b>														
Size 5 to 12	–	–	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	–	–	–
Size 16 to 65	–	–	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,1
<b>RADEX-NC®</b>														
EK and DK	–	–	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,0	1,1

Operating factor $S_B$		
<b>ROTEX® GS*</b>		
<b>Backlash-free drives</b>		
Main spindle drive of machine tool	2,0 – 5,0	
Positioning drives		
Ball screw drive/toothed belt drive	3,0 – 5,0	
Gearbox	$i \leq 5$	8,0
	$i > 5 - \leq 7$	5,0
	$i > 7$	3,0
<b>Servo-hydraulic drives</b>		
With pulsating load <sup>1)</sup>	1,0 – 1,2	
With alternating load <sup>2)</sup>	1,3 – 1,5	
<b>TOOLFLEX®, RADEX®-NC</b>		
Uniform motion	1,5	
Nonuniform motion	2,0	
Shocking motion	2,5 – 4,0	
Für Antriebe an Werkzeugmaschinen (Servomotoren) sind Werte von 1,5 – 2,0 einzusetzen		

Shock factor $S_A/S_L$	
<b>Main spindle drive</b>	
Moderate shocks	1,0
Average shocks	1,4
Heavy shocks	1,8
Positioning drive <sup>3)</sup>	
< 60	1,0
≥ 60 – < 300	1,4
≥ 300	1,8

\*When using the spider 64 Sh-D-GS or 72 Sh-D-GS a factor of at least 4 or steel hubs have to be considered.

<sup>1)</sup> With pulsating load the use of aluminium is permissible.

<sup>2)</sup> With alternating load please make use of steel hubs.

<sup>3)</sup> Starts per minute

Shaft encoder drives: Subject to the low torques to be transmitted the coupling size for shaft encoder drives is selected according to the shaft diameters to be connected.

## Coupling selection

The coupling selection of the backlash-free servo couplings is based on DIN 740 part 2, but with specific factors. The coupling has to be dimensioned in a way that the permissible coupling load is not exceeded during any operating condition. For this purpose the actual loads have to be compared to the permissible parameters of the coupling. The shaft-hub-connection has to be investigated by the customer. The size of the coupling must be selected so that the following conditions are met.

### 1. Backlash-free drives

$$T_{KN} \geq T_N \cdot S_t \cdot S_B$$

and

$$T_{KN} \geq T_S \cdot S_t \cdot S_B$$

In case of a loaded torque:  $T_{KN} \geq T_S \cdot S_t \cdot S_B + T_N \cdot S_t$

Taking into account the temperature factor  $S_t$  and the operating factor  $S_B$ , the permissible rated torque  $T_{KN}$  must be at least as big as the the rated torque  $T_N$  of the machine. Furthermore the permissible rated torque  $T_{KN}$  has to be at least as big as the maximum driving torque, even under the influence of the temperature factor  $S_t$ .

Calculation of the maximum driving torque  $T_S$ :

$$\text{Shock loads on driving side } T_S = T_{AS} \cdot M_A \cdot S_A \quad \longrightarrow \quad M_A = \frac{J_L}{(J_A + J_L)}$$

$$\text{Shock loads on loaded side } T_S = T_{LS} \cdot M_L \cdot S_L \quad \longrightarrow \quad M_L = \frac{J_A}{(J_A + J_L)}$$

### 2. Servo-hydraulic drives

$$T_{KN} \geq T_{AS} \cdot S_t \cdot S_B$$

Taking into account the ambient temperature and the operating factor, the permissible rated torque  $T_{KN}$  of the coupling has to correspond at least to the peak torque of the driving side  $T_{AS}$ .

### Please note:

For general applications (not backlash-free applications) please follow coupling selection according to DIN 740 part 2 (page 10 et seqq.)

# COUPLING SELECTION ACC. TO DIN 740 PART II WITH SPECIFIC FACTORS

## Example of calculation for positioning drives

Requested: Backlash-free coupling damping vibrations

→ ROTEX® GS

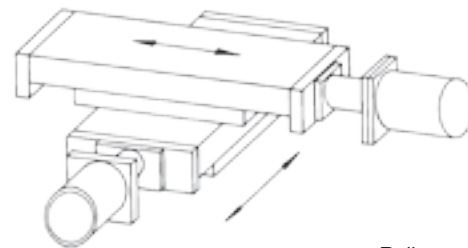
Application: Connecting servo motor and ball screw drive for backlash-free positioning

→ Coupling selection following page 19, item 1: Backlash-free drives

### Given: Details of driving side

Servo motor

Rated torque  $T_{AN}$ : 43 Nm  
 Max. driving torque  $T_{AS}$ : 144 Nm  
 Moment of inertia  $J_{Mot}$ : 0,0108 kgm<sup>2</sup>  
 Diameter of motor shaft: 32 k6 without feather keyway



Ball screw

Ambient temperature: 40 °C →  $S_t = 1,2$  (see page 19)  
 Starts per minute: 60 →  $S_A = 1,0$  (see page 19)

### Given: Details of machine on driving side

Ball spindle  $J_{Sp}$ : 0,0038 kgm<sup>2</sup>  
 Spindle pitch s: 10 mm  
 Diameter of spindle shaft: 30 k6 without feather keyway  
 Mass of slide + work piece  $m_{SI}$ : 1030 kg  
 No load torque available

Required: hohe Drehsteifigkeit →  $S_B = 4$  (s. Seite 19)

### Calculation

#### 1. Backlash-free drives

- Loading by rated torque (pre-selection)

$$T_{KN} \geq T_{AN} \cdot S_t \cdot S_B \quad \rightarrow 43 \text{ Nm} \cdot 1,2 \cdot 4 \quad \rightarrow T_{KN} \geq 206,4 \text{ Nm}$$

- Coupling selection (pre-selection)

ROTEX® GS 38

Spider 98 Shore-A with clamping ring hubs 6.0 light:

Mass moments of inertia (see page 130)

$T_{KN} = 325 \text{ Nm}$

$J_{KA} = 0,000517 \text{ kgm}^2$

$T_{K \text{ max.}} = 650 \text{ Nm}$

$J_{KL} = 0,000517 \text{ kgm}^2$

- Load by maximum driving torque, not including load torque

$$T_{KN} \geq T_S \cdot S_t \cdot S_B$$

$$\text{Shock on driving side } T_S = T_{AS} \cdot M_A \cdot S_A \quad \rightarrow = 144 \text{ Nm} \cdot 0,379 \cdot 1,0 \quad \rightarrow T_S = 54,58 \text{ Nm}$$

$$M_A = \frac{J_L}{(J_A + J_L)} \quad \rightarrow = \frac{0,006917 \text{ kgm}^2}{(0,011317 \text{ kgm}^2 + 0,006917 \text{ kgm}^2)} \quad \rightarrow M_A = 0,379$$

$$J_A = J_{Mot} + J_{KL} \quad \rightarrow 0,0108 \text{ kgm}^2 + 0,000517 \text{ kgm}^2 \quad \rightarrow J_A = 0,011317 \text{ kgm}^2$$

$$J_L = J_{Sp} + J_{SI} + J_{KL} \quad \rightarrow 0,0038 \text{ kgm}^2 + 0,0026 \text{ kgm}^2 + 0,000517 \text{ kgm}^2 \quad \rightarrow J_L = 0,006917 \text{ kgm}^2$$

$$J_{SI} = m_{SI} \cdot \left(\frac{s}{2 \cdot \pi}\right)^2 \quad \rightarrow 1030 \text{ kg} \cdot \left(\frac{0,01}{2 \cdot \pi}\right)^2 \quad \rightarrow J_{SI} = 0,0026 \text{ kgm}^2$$

$$\rightarrow T_{KN} \geq 54,58 \text{ Nm} \cdot 1,2 \cdot 4 \quad \rightarrow T_{KN} \geq 261,9 \text{ Nm}$$

$T_{KN}$  with 325 Nm  $\geq 261,9 \text{ Nm}$

- Review of shaft-hub-connection: Friction torque for clamping ring hubs type 6.0 light

The coupling has to be dimensioned such that the permissible friction torque is not exceeded during any operating condition.

$$T_R \geq T_{AS} \quad \text{values } T_R \text{ see page 130}$$

Friction torque of ROTEX® GS 38 clamping ring hub 6.0 light Ø30 H7/k6  $T_R = 443 \text{ Nm} > 144 \text{ Nm}$

### Result

The coupling is sufficiently dimensioned.

# COUPLING SELECTION ACC. TO DIN 740 PART II WITH SPECIFIC FACTORS

## Example of calculation for main spindle drives

**Requested:** Backlash-free, axial plug-in coupling for high speeds → ROTEX® GS  
**Application:** Connecting servo motor and main spindle in a grinding machine  
 Coupling selection following page 19, item 1: Backlash-free drives

### Given: Details of driving side

Servo motor

Rated torque with operation  $T_{AN}$ : 154 Nm  
 Max. driving torque  $T_{AS}$ : 190 Nm  
 Max. speed: 6000 1/min  
 Moment of inertia  $J_{Mot}$ : 0,316 kgm<sup>2</sup>  
 Diameter of motor shaft: 30 k6 without feather key-way  
 Ambient temperature: 60 °C →  $S_t = 1,4$  (see page 19)  
 Shock factor  $S_A$ : moderate shocks →  $S_A = 1,0$  (see page 19)

### Given: Details of machine on driving side

Moment of inertia of load side  $J_{HS}$ : 0,1094 kgm<sup>2</sup>  
 Diameter of main spindle shaft: 30 k6 without feather key-way  
 No load torque available

Required: no high torsion stiffness →  $S_B = 2$  (see page 19)

### Calculation

#### 1. Backlash-free drives

● Loading by rated torque (pre-selection)  $T_{KN} \geq T_{AN} \cdot S_t \cdot S_B$  →  $154 \text{ Nm} \cdot 1,4 \cdot 2$  →  $T_{KN} \geq 431,2 \text{ Nm}$

● Coupling selection (pre-selection)

ROTEX® GS 42

Spider 98 Shore-A with clamping ring hubs 6.0 light:

Mass moments of inertia of page 130

$T_{KN} = 450 \text{ Nm}$

$J_{KA} = 0,001117 \text{ kgm}^2$

$T_{K \text{ max.}} = 900 \text{ Nm}$

$J_{KL} = 0,001117 \text{ kgm}^2$

● Load by maximum driving torque, not including load torque

$T_{KN} \geq T_S \cdot S_t \cdot S_B$

Shock on driving side  $T_S = T_{AS} \cdot M_A \cdot S_A$  →  $144 \text{ Nm} \cdot 0,376 \cdot 1,0$  →  $T_S = 54,14 \text{ Nm}$

$M_A = \frac{J_L}{(J_A + J_L)}$  →  $= \frac{0,191517 \text{ kgm}^2}{(0,317117 \text{ kgm}^2 + 0,191517 \text{ kgm}^2)}$  →  $M_A = 0,376$

$J_A = J_{Mot} + J_{KL}$  →  $0,316 \text{ kgm}^2 + 0,001117 \text{ kgm}^2$  →  $J_A = 0,317117 \text{ kgm}^2$

$J_L = J_{MS} + J_{KL}$  →  $0,1094 \text{ kgm}^2 + 0,001117 \text{ kgm}^2$  →  $J_L = 0,191517 \text{ kgm}^2$

$T_{KN} \geq 54,14 \text{ Nm} \cdot 1,4 \cdot 2$  →  $T_{KN} \geq 151,6 \text{ Nm}$

$T_{KN}$  with 450 Nm  $\geq 151,6 \text{ Nm}$

● Review of shaft-hub-connection: Friction torque for clamping ring hubs type 6.0 light

The coupling has to be dimensioned such that the permissible friction torque is not exceeded during any operating condition.

$T_R \geq T_{AS}$  values  $T_R$  see page 130

Friction torque of ROTEX® GS 42 clamping ring hub 6.0 light Ø30 H7/k6  $T_R = 507 \text{ Nm} > 190 \text{ Nm}$

### Result

The coupling is sufficiently dimensioned.